

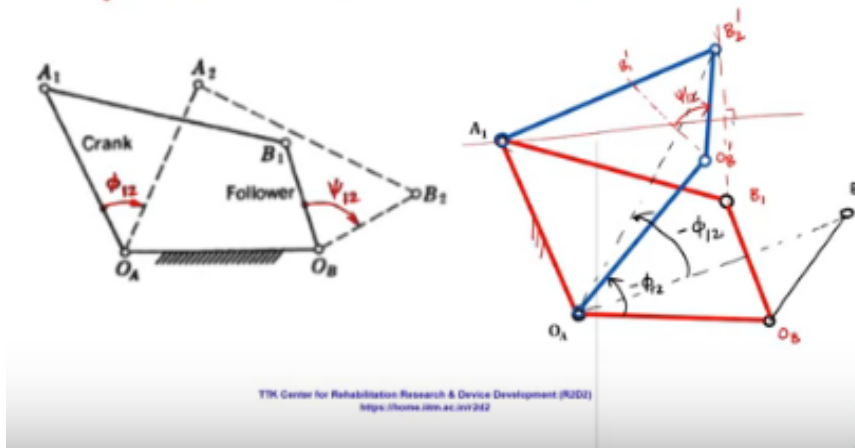
Theory of Mechanisms
Lecture - 9
Function Generation

So,

Refer slide time: (0:14)

Function generation

Getting a specified rotary output from a given rotary input

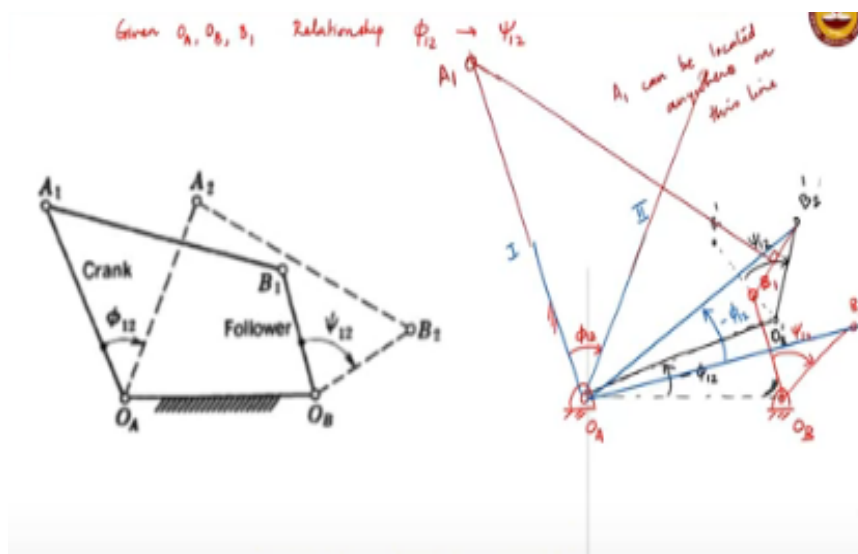


We started looking at, function Generation, synthesis for, function generation, where essentially we are looking at getting, a specified, rotary output, from a given rotary input. Okay? So if I have a crank in the follower like this, giving an input, of ϕ_{12} , to the input link, gets me a specified angular output. Okay? So if I want. So we will first look at, what this looks like. So if I have a linkage, already designed, that does this. Okay? What is it? What is the relationship that I'm going to look for, that will help me in synthesizing it? Okay? So that is what I am looking at here. So I have O_A and O_B , this is my fixed link initially. And again, we look at inversion. Okay? I say instead of this being my fixed link. Okay? I look at the case where, this link is my fixed link. Okay? I change it; I changed my fixed link, to the input link, because my relative motions, among the links, have to be the same, the relationships among them have to be the same. So if I have this configuration, then regardless of how I, regardless of which link is fixed, this would be the relationship, among the four links. Okay? So we are using the principle of inversion. So if this is, a 1, B_1 , then I can look at this as. Okay, if now this is the fixed link, O_A , O_B , has to move by, minus ϕ_{12} . Okay? Move in the other direction. And then if this is the original position, of B_1 , you know, say, I maintain, this move it, rotate the whole thing, O_A , O_B , B_1 and then I have to, move this again by, ϕ_{12} , to achieve my final position. Okay? So if this linkage is designed for ϕ_{12} , giving me ψ_{12} , then this would be the relationship, between the two configurations, if I sit on the crank and observe the motion by inversion. So that means, this would be, the position of B_2 dash. So B_2 as observed, from the fixed crank. Okay? So I call that B_2 dash.

And what this tells me is this link if I look at O_B , b_1 . Okay? This is O_B dash; O_B has moved to O_B dash, b_1 has moved to b_2 dash. Right? This okay. This should be, yeah this, this is B , in relation to, with just this rotation. So let's call this b_1 dash and then it moves to B_2 dash, b_2 dash should be its final position. Okay? So if I look at this, then, because of the fact that I'm observing it from the crank. Okay? That means, I have moved, so if I look at this link, O_B , b_1 ? O_B , b_1 , has moved from O_B , b_1 , to O_B dash, B_2 dash. So that is essentially motion generation with respect to, this is a fixed link. O_B , b_1 has moved to O_B dash, B_2 dash, which means, as observed from here, O_A would be the point, about which, B_1 moves along an arc.

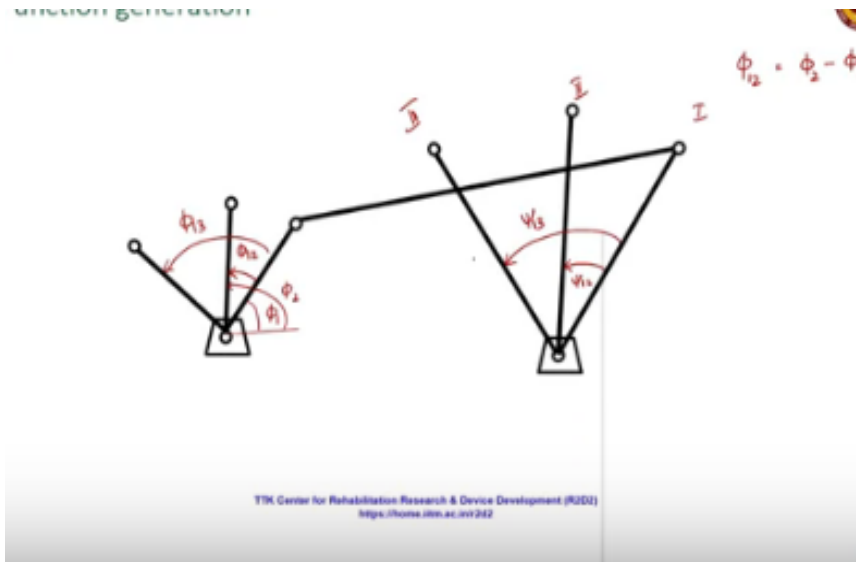
Which means, a_1 essentially lies on, the perpendicular bisector of b_1, b_2 dash. So this would be the perpendicular bisector of b_1, b_2 dash. So this should be the relationship, between the various points, in the linkage, if this is satisfied. Okay? Now we are going to go back, okay, this is assuming, I have a linkage, which does it, I am looking at, what the geometry will be like, now I am going to Essentially use this to design the linkage. Okay? So suppose you have, I'm given this, I want to synthesize, a linkage that will do this. Okay? Say I'm given O_A, O_B , I am given the location of the fixed pivots and I have the link of, the link length for the input link specified, so a_1 is specified. Now I want to synthesize the linkage, such that, it has this relationship. So I am given, O_A, O_B, A_1 , required relationship is, between Φ_{12}, Ψ_{12} , Φ moving from 1 to 2, should give me, Ψ_{12} . Okay? So now, if I look at this. Okay, let's do this. Suppose I instead of, I'll tell you why, but now. Let's, let's just do this, okay, fine a 1, is, it'll be the opposite of what we did there, but that's okay. Okay, so let's say, I don't know where b_1 should be Located, but let's say this is, this angle would be. What would it be? Ψ_{12} . Okay? Between two positions, this should be the angular relationship. So in this case, I know A_1 . So let me sit on position one. So I sit on the link about which I don't have enough information. Because my vantage point, okay the point from which I am going to observe, is going to be O_B . Okay? Let's say. Okay, so I have if I'm sitting on position one, so let's say this is my fixed link now. Okay? So I have to move, so if I if you look back at this previous one, you can see that, essentially, if I do these two rotations, that's the same as, this is fine, so I have B_2 here. Okay? So if I look at O_A, O_B, B_2 , okay? If I rotate O_A, O_B , by minus Φ_{12} and this by, Ψ_{12} , that's the same as rotating this line $O_A B_2$ to by this angle, which is again, minus Φ_{12} . Do you see that? This triangle it's, it's not a very good representation of it, but essentially, because this is a rigid body, now. Okay? I'm taking the second position, this is where the location should be, so rotating that by the angle, by which this whole thing should move, is, gives me B_2 dash, that's what I'm going to use here. So instead of doing this two-step, rotation, by minus Φ_{12} and then Ψ_{12} , I can instead, straightaway take, the location of the second position and rotate that by minus Φ_{12} .

Refer slide time: (10:28)



So in this case now, let me say that, okay, I have OA, I have OB. Okay? And I am given B1, so I'm given OA, OB and B1. Okay? And, relationship, Φ_{12} , gives me s_{I12} , in the output so this is, so I know this. Okay? It's a two position synthesis, so let's just say, I have two positions of the input link, I know the angle should be, Φ_{12} . Okay? This is also given, so let me just mark that in red. But I don't know, where exactly, so my synthesis is basically, to determine my crank length and my coupler length. Okay? Because I have an output link length, in the rocker length, I have the fixed link length; I want to find the input link length and the coupler link. If I find the pivot a, it gives me both of those. Okay? Because OA, A, will be my input link length, I know where B is, so AB will give me my coupler length. Okay? So now, I sit on instead of OA, OB, being my fixed link, I sit on link one instead, sorry, on position one and I say I'm going to rotate OA, OB and this to meet me at the new place. Okay? So let's just say, in the second position, OA, OB, this is closer to OA, OB, by this Angle, Φ_{12} . Right? In the second position, OA, A2, is rotated by Φ_{12} , towards OA, OB. So my first task here would be to, rotate OA, OB, by, bring it closer to this, by minus Φ_{12} . That is OA, OB. So this is OB dash, just do it in a, two step process. Okay? Then, in the second position, if this is, when I rotate, assume I don't change the inclination of OB, B1, first, I keep it as it was in the first position, so assume this is like welded right now, I rotate it, rotate the whole thing, as a rigid body, by minus Φ_{12} , so this is B1 dash. Now, but in the second position, I want OB, B1, OB, B2, to be actually rotated by s_{y12} , with respect to OB, B1. So now this should be my, B2 dash. This angle should be s_{I12} . Okay? This is the same, doing this two-step process, this, Plus this, is the same, as, if I get, if I take OA to B2. Okay? So if you look at this, OA, OB, B2, that forms a, rigid triangle, so whatever OA, OB, rotates by, in the second position, is what this would also rotate by. So that is the same as, rotating, so this angle will be, nothing but, minus Φ_{12} . You see that? So you can instead of doing it as a two-step process, you can directly take, B2 to be B2 dash. Okay? So, the movement from b1 to b2 dash, b1 was in the first position, b2 dash is what you observed, from the inversion, that movement happened about a point a, on the input link. Okay? Such that, it moved in a circular path, so this, to this, so the pivot a, lies somewhere, on the perpendicular bisector, of that. So it could be anywhere on the perpendicular bisector, of this, because again my vantage point, is only OA, I am only, this movement happens about, this point OA. So if A can be located, A1 can be located anywhere on this line. One possibility is where it intersects one or you know? Or somewhere here, then this would give me, A1. If I do it to intersect, this line, then, this would give me A1, in this position itself. Okay? You can see, how. This should be my A1, then A2 would be on this, so when that moves by Φ_{12} , I will have B1 move to B2. Is that clear now? So it's a two-step rotation actually, the shortcut is to just rotate, that line. Okay? And depending on where you are sitting, that angle is what you have to take into account. You have to rotate in the opposite direction, of that angle in order, to get the new position. Okay?

Refer slide time: (18:32)



You can do the same thing for three positions. So you have, see what the angles, so I have say Phi 12, position 1 to 2, then I have phi13 and I have 1, 2, 3 here, 1. So this would be s I 1 2, this would be s I 1 3. So when I say, say if this is, some angle Phi 1 and this is Phi 2, Phi 1 2 is, essentially Phi 2 minus Phi 1. That's the notation that is used. So this Is, s I 1 3. So let's go through the steps for that.

Refer slide time: (19:44)

Function generation

- ① Sit on the follower in the 1st configuration
- ② Same relative movement from config I to II is obtained by rotating the frame by $-\psi_{12}$ (same as rotating ${}_{O_B}A_2$ by $-\psi_{12}$ about O_B). This gives A_2'
- ③ Similarly, A_3' is obtained by rotating ${}_{O_B}A_3$ by $-\psi_{13}$ about O_B
- ④ B_1 lies at the centre of the circle passing through 1.

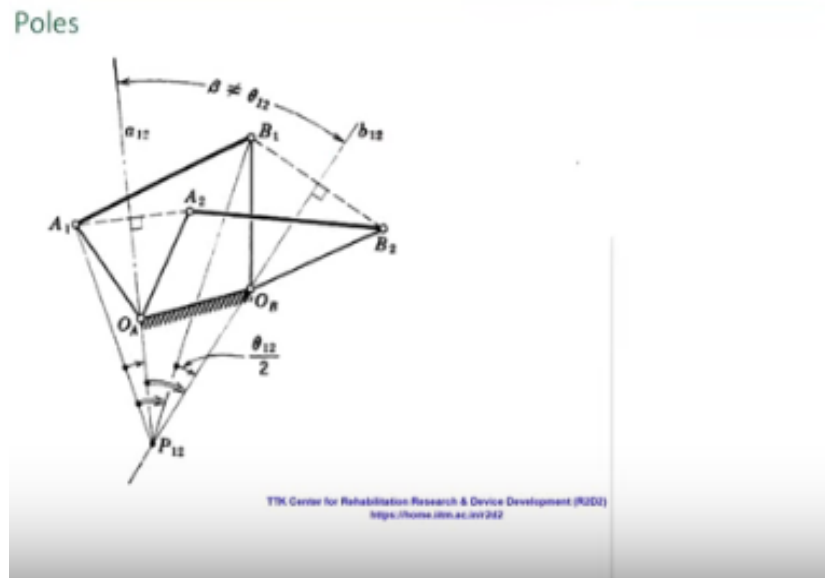
TUM Center for Rehabilitation Research & Device Development (R2D2)
<https://home.tum.ac.jp/242>

87

This one I came prepared. So let's say, I'm given this, I have ϕ_{12} , this would be ϕ_{23} , I can look at it and I want s_{I12} , s_{I13} , or s_{I23} . Whichever two of the three, so this is what is given, given OA, OB, I am given, I am giving you a crank length, so I'm locating a 1, somewhere on that line. So first, I take OB, A2. What will I rotate it by now? Yeah. So I rotate to, A2 dash, by, minus s_{I12} . Okay? Then A1, A2 dash, so it lies somewhere on that perpendicular bisector, but here I have one more condition, so I'm not ready to pick, that yet. Okay? Second step or third, OB, A3, that's in the third position. What will I rotate it by? Minus ϕ_{13} . So this angle, minus s_{I13} . Okay? Now perpendicular bisector, from where to where? A12, A3 dash, where they intersect, gives me, b1, yeah. That completes my linkage. Because I sat on position one and use the angles for position two and three, I draw it, this gives me B1. If I sat on position two and then I transformed, the location of a1 and a3, then what I would get? Is the linkage in the second configuration? So the B that I get there will be B2. Okay? So this is the, construction you will follow. Again it doesn't matter, if I give you B1, you will do the opposite of this. So sit on the link, where you have less information, because you will need the location of the other link, after the rotation that is what this one. Okay this one is clear? This problem statement okay, I have OA, A1, rotating to, OA, A2 and this thing. And the output link, when this rotates by ϕ_{12} , this should rotate s_{I12} , similarly that. I have information about, the length of the crank. Okay. I'm given. So I know the location of A1, location of a2 and a3. Okay? So I sit, on essentially the first position. Okay? But with the vantage point OB. Okay? I sit at and I know that, OB, A2, should be my relative location, but with the whole thing, rotated by, so this is OB A2. Okay? What, the location of A2, would be in the second position and if I'm not moving from the first position, I essentially have to, rotate. So to move to the second position, I would have moved s_{I12} , but I am not moving ϕ_{12} , so I had to rotate this, by minus ϕ_{12} . I am staying in one inversion. Inversion means I am staying that is now my fixed link, I am sitting on the follower, on the first, at the first position. So I am rotating this, by ϕ_{12} , which gives me the new location, for it. Okay? Relative positions all remain the same. For the third position, B lies somewhere on this perpendicular bisector. Because from the point B, the pivot point, that's my fixed pivot, now. Okay? That's my fixed pivot, from by inversion, the movement from a1 to a2 dash, occurs about a circular arc, that is centered at, b1. So my b1 lies somewhere on that perpendicular bisector. Same thing for the third position I join OB, a3 and rotate that, I'll write down some of this, if you want. OB, a3, rotates that, by the angle, minus s_{I13} , to get a 3 dash. Now again a 3 dash, also lies on the same circle, that a 2 dash will lie. Okay? So, I can draw the perpendicular bisector, I could have also drawn the perpendicular bisector, of a2, a3 dash. It will still intersect at this point. Essentially a1, a2 dash, a3 dash, will lay on a circle, centered at, b1. Okay? Because by inversion, when this is my fixed link, the other links, OA and A1 will move on circular arcs, essential. Link 1 and here, you mean here, yeah. Because that would be too restrictive you won't be able to find a solution. Okay? So because this is a unique point now in the previous case I. I can still find a solution. But in the previous case, anything on the perpendicular bisector, would still work. So in the previous case, you have infinity of solutions, for the two position problem, in function generation. In the 3 position case, once you determine, a 1, once you fix a 1, you only have a unique solution, again as in the case with the 3 position motion generation, if I choose as different a1. Okay? If I choose, some other thing as my Length, I can get more solutions. Okay? So I could essentially have anything in the plane, because, if I am only concerned with angular displacements, three-position and function generation means, two angular displacements. 1 to 2 and 2 to 3. Okay? So I could actually have, more solutions, if I change where I pick A1. So I have an in, in the previous case, for the two position function generation, I have infinity, to the power three, infinity cube solutions. Because I can I have infinity square, for the location of, you know for the length, of one of the links, I can pick any point in the plane and any point on the Perpendicular bisector. So that is infinity of solution, Infinity square, into infinity. Here, once I choose A1, which is infinity square, choices, I only have one solution. Okay? So, I'll just sit on the follower,

in the first configuration, same relative movement, from config, 1 to 2, is obtained by, rotating the frame, by minus $s I 12$, which is the same as, rotating, $OB, A2$, by minus $s I 12$, about OB . This gives $A2$ dash. Similarly, $A3$ dash is obtained by, rotating the frame, or $OB, A3$, by minus $s I 13$, about OB . $B1$ lies at the center of the circle, passing through, $A1, A2$ dash, $A3$ dash. Okay?

Refer slide time: (31:38)



So for function Generation, there's a method using relative poles also, Instead of inversion. This is based on the properties of the pole that we saw in the one of the earlier classes. We will use that for function generation, in the next class.